

DESTABILIZING EFFECT OF SMALL DAMPING

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In 1890, Lord Kelvin, [6], indicated that the stability or instability of the equilibrium state of a system without dissipative or gyroscopic forces is not changed by the introduction of damping. This need not be true when dissipative forces are applied to other than non-gyroscopic, conservative systems. In fact, Ziegler, [7, 8, 9], proved that a critical load of a non-conservative system can be decreased by adding damping. Further studies on this subject have been done by Bolotin, [1], Herrmann, [4], Hagedorn, [3], and others.

It is our purpose to show this effect in the mechanical model of fig. 1. It consists of two rigid weightless bars of equal length ℓ , carrying concentrated masses $m_1 = m_2 = m$ at the endpoints and connected by a linear torsional spring. The lower endpoint of the linkage is kept fixed and angular motion is resisted by another torsional spring. Friction is present in the system at the pivots.

After a suitable change of variables, the differential equations of motion, which can be derived using Lagrange equations, are:

$$(1) \quad 2x'' + y'' \cos(x-y) + (y')^2 \sin(x-y) + (b_1 + b_2)x' - b_2y' + 2x - y - \lambda \sin(x-y) = 0$$

$$(2) \quad x'' \cos(x-y) + y'' - (x')^2 \sin(x-y) - b_2x' + b_2y' - x + y = 0$$

$$(\quad)' = \frac{d}{dt}$$

$x(t) = y(t) = 0$ is the only time independent solution of (1) and (2). Since (2) implies $x = y$ and then (1) implies $x = 0$.

In order to study the stability of this equilibrium state, which we shall denote by X_0 , we consider the variational problem of (1) and (2) with respect to X_0 :

$$(3) \quad 2x'' + y'' + (b_1 + b_2)x' - b_2y' + (2 - \lambda)x - (1 - \lambda)y = 0$$

$$(4) \quad x'' + y'' - b_2x' + b_2y' - x + y = 0$$

We investigate solutions of the variational problem in the form $x = a_1 e^{\sigma t}$, $y = a_2 e^{\sigma t}$. This leads to the homogeneous system of algebraic equations:

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$$(5) [2\sigma^2 + (b_1 + b_2)\sigma + 2 - \lambda]a_1 + [\sigma^2 - b_2\sigma + \lambda - 1]a_2 = 0$$

$$(6) [\sigma^2 - b_2\sigma - 1]a_1 + [\sigma^2 + b_2\sigma + 1]a_2 = 0$$

and to the characteristic equation:

$$(7) \sigma^4 + (b_1 + 5b_2)\sigma^3 + (6 + b_1b_2 - 2\lambda)\sigma^2 + (b_1 + b_2)\sigma + 1 = 0$$

If $b_1 = b_2 = 0$, (no damping), the four roots of (7) are

$$(8) \sigma_j = \pm [\lambda - 3 \pm \sqrt{(\lambda - 2)(\lambda - 4)}]^{1/2}$$

These roots are pure imaginary for λ less than the critical load $\lambda_c^u = 2$ taking on the values $\pm i$ at it. Thus, X_0 is neutrally stable for λ on the interval $[0, 2]$. For $\lambda > \lambda_c^u$ two of the roots have positive real part, then X_0 is linearly unstable, it is then possible to prove that X_0 is nonlinearly unstable for $\lambda > \lambda_c^u$. For $\lambda \leq \lambda_c^u$, the nonlinear stability or instability of X_0 can not be decided by the nature of the solutions of the variational equations as was pointed out by Liapunov, [5]. It has not been determined if X_0 is asymptotically stable or not in this case.

When the damping coefficients are not simultaneously zero, application of the Routh-Hurwitz criterion shows that there exists a critical load λ_c , defined by

$$(9) \lambda_c = 2 \frac{b_1^2 + 6b_1b_2 + b_2^2}{b_1^2 + 6b_1b_2 + 5b_2^2} + \frac{b_1b_2}{2}$$

such that the real parts of the four roots of (7) are negative if $\lambda < \lambda_c$. At $\lambda = \lambda_c$, the real parts of two of the roots vanish, becoming positive for $\lambda > \lambda_c$. Hence, X_0 is asymptotically stable for $\lambda < \lambda_c$ and nonlinearly unstable for $\lambda > \lambda_c$. At $\lambda = \lambda_c$ the solution X_0 is linearly neutrally stable.

Applying the argument theorem of analytic functions, it is possible to prove that two of the roots of (7), let us say σ_3 and σ_4 have negative real parts for every λ .

The behavior of the real and imaginary parts of the roots as functions of λ is displayed in figures 2a, b, c and 3a, b, c for some representative values of b_1 and b_2 .

we have then seen that the critical force is $\lambda_c^u = 2$ when $b_1 = b_2 = 0$. In the damped case, and when $b_2 = 0$, the critical load and $\sigma_1(\lambda_c)$, which are equal to 2 and i respectively, are independent of b_1 , while for $b_2 \neq 0$, λ_c depends on the ratio as well as on the magnitudes of the damping coefficients. The complex frequency

$$(10) \quad \sigma_1(\lambda_c) = i\omega_0 = i \frac{b_1 + b_2}{b_1 + 5b_2}$$

depends only on the ratio of the damping coefficients.

In the limiting case of vanishing damping, i.e. when b_1 and b_2 tend simultaneously to zero, the results corresponding to the undamped model can not be obtained unless $\lim (b_1/b_2) = \infty$ as $b_1, b_2 \rightarrow 0$.

Differentiating λ_c with respect to b_2 and equating to zero, we have:

$$(11) \quad b_1 b_2^4 \left[\frac{b_1^4 + 4b_1(3b_1^2 - 8)b_2 + 2(23b_1^2 - 48)b_2^2 + 60b_1b_2^3 + 25b_2^4}{(b_1^2 + 6b_1b_2 + 5b_2^2)^2} \right] = 0$$

A root of this equation is $b_2 = 0$. In addition we have determined numerically that it also has two positive real roots, let us say $b_{21}(b_1)$ and $b_{22}(b_1)$, for fixed b_1 in the interval $0 < b_1 < \bar{b}_1$, $\bar{b}_1 = 1.334418$.

For fixed b_1 in this interval, λ_c has a relative maximum at $b_{21}(b_1)$ and a relative minimum at $b_{22}(b_1)$, (Table I). This behavior is illustrated in fig. 4 for which we have chosen an arbitrary value of b_1 in $0 < b_1 < \bar{b}_1$.

At any point (b_1, b_2) with b_1 in the interval $(0, \bar{b}_1)$ and $b_{21}(b_1) < b_2 < b_{22}(b_1)$, the damping has a destabilizing effect since λ_c is a monotone decreasing function of b_2 and therefore the critical force can be lowered by incrementing the damping in the upper joint of the mechanical model.

As a consequence of the discontinuity of the critical force, in the limit case of vanishing damping, the analysis of the destabilizing effect of small damping requires a special treatment. We have found numerically that as b_1 tends to zero, so does $b_{21}(b_1)$. Moreover, $db_{21}(b_1)/db_1$, which can be found by implicit differentiation of the left hand side of equation (11) and evaluated numerically, tends to zero too.

by the theory and the experimental measures which do not reflect the

destabilizing effect. He solves the discrepancy by considering external

friction. Another possible explanation has been given by Dimentberg, [2],

who conjectures that in a more realistic model, hysteresis effects should

By equating λ_c to 2, one obtains:

$$(12) \quad 0.5 b_2 [5 b_1 b_2^2 + 2(3 b_1^2 - 8) b_2 + b_1^3] = 0$$

A solution of this equation is $b_2 = 0$, and the quadratic factor yields two other roots when solved for b_2 in terms of b_1 . Those two roots, which we shall denote by $b_{23}(b_1)$ and $b_{24}(b_1)$, (fig. 5), are real and positive provided $0 < b_1 < b_1^0 \equiv \sqrt{6 - 2\sqrt{5}} \simeq 1.236068$, and therefore (12) has three real roots in an interval which is contained in the interval where (11) has two positive real roots. On $0 < b_1 < b_1^0$, $b_{23}(b_1)$ is a monotone increasing function and its limit, as well as the limit of its derivative with respect to b_1 , is zero as b_1 tends to zero. By Rolle's theorem, $0 < b_{21}(b_1) < b_{23}(b_1)$ on the interval where they are both defined, (Table I).

This indicates that if in the undamped case, X_0 is proven to be nonlinearly stable for $0 < \lambda \leq 2$ then, by choosing the damping coefficients in the ranges $0 < b_1 < b_1^0$ and $b_{23}(b_1) < b_2 < b_{24}(b_1)$, that the critical force decreases, i.e., $\lambda_c < 2$. Thus damping can destabilize the system. That is the critical force can be lowered in the presence of damping. On the other hand, if $0 < b_2 < b_{23}(b_1)$, then $\lambda_c > \lambda_c^u$ and damping has a stabilizing effect in that region.

Ziegler, [10], has shown that this phenomenon can also be found in a model consisting of a disk mounted in a shaft rotating with angular velocity ω , when only internal damping is considered. He points out that there exist differences between the critical velocity predicted by the theory and the experimental measures which do not reflect the destabilizing effect. He solves the discrepancy by considering external friction. Another possible explanation has been given by Dimentberg, [2], who conjectures that in a more realistic model, hysteresis effects should

be considered. Other examples of destabilizing effect of damping can be found in Ziegler, [10].

Herrmann, [4], has studied the destabilizing phenomenon in a mechanical model like the one in fig. 1, with $m_1 = 2m$, $m_2 = m$, finding $\lambda_c^u = 3.5 - \sqrt{2}$ and

$$(13) \quad \lambda_c = \frac{4 b_1^2 + 33 b_1 b_2 + 4 b_2^2}{2(b_1^2 + 7 b_1 b_2 + 6 b_2^2)} + \frac{b_1 b_2}{2}$$

Assuming $b_i \ll 1$, $i = 1, 2$, and defining the critical force for small damping as

$$(14) \quad F_d = \frac{4 b_1^2 + 33 b_1 b_2 + 4 b_2^2}{2(b_1^2 + 7 b_1 b_2 + 6 b_2^2)}$$

he neglected $b_1 b_2 / 2$ and considered the ratio F_d / λ_c^u as a function of

$\beta = b_1 / b_2$. The forces ratio is strictly less than one, except when $\beta = \beta^* = 4 + 5\sqrt{2}$ where it is equal to 1. In this way he concluded that

the presence of damping has a destabilizing effect which is eliminated only at that particular value β^* . This is in contradiction with our

result that damping has a stabilizing effect on an open region of

the b_1 - b_2 plane instead of only on a straight line. To see that Herrmann's conclusion is erroneous for (13) we consider the ratio λ_c / λ_c^u ,

(Herrmann's functions λ_c and λ_c^u), instead of F_d / λ_c^u and do not neglect

$b_1 b_2$ compared to F_d . We numerically determine that λ_c / λ_c^u has the

value $4 / (7 - 2\sqrt{2}) \simeq .959$ at $b_2 = 0$, it increases to a relative

maximum, decreases with increasing b_2 up to a point where it reaches a

relative minimum and finally, increases, all this provided $0 < b_1 < 1.126$,

i.e., the same type of behavior our function λ_c has, (fig. 6).

Within this interval, there are three different values of b_2 , let us say b_{21}, b_{22}, b_{23} , all of them depending on b_1 , where the ratio λ_c / λ_c^u takes on the value 1. Furthermore, b_{21} and b_{22} are differentiable functions of b_1 and $\lim b_{21} = \lim b_{22} = 0$ as b_1 tends to zero. In Table II are given some numerical values of b_{2i} , $i=1,2,3$ for small b_1 , as well as the inverses of the derivatives of b_{21} and b_{22} with respect to b_1 ; it follows from those numerical results that both inverses approach the common value β^* .

These results can also be found analytically. First, it must be noted that $b_1 b_2 / 2$ can not be neglected when λ_c is compared with λ_c^u since their difference is equal to

$$(15) \quad \lambda_c - \lambda_c^u = \frac{1}{2} b_1 b_2 - (3 - 2\sqrt{2}) \frac{[b_1 - \beta^* b_2]^2}{2(b_1^2 + 7b_1 b_2 + 6b_2^2)} \\ = \frac{1}{2} b_1 b_2 + (F_d - \lambda_c^u)$$

and both terms can have the same order of magnitude when b_1 and b_2 are small and $b_1 \simeq \beta^* b_2$. Moreover, if it neglected, one is misled to the result $b_1 = \beta^* b_2$, while this is only the first order approximation to two different positive real roots of

$$(16) \quad \lambda_c - \lambda_c^u = 0$$

In order to find a better approximation, let us set

$$b_1 = \epsilon \cos \alpha(\epsilon) \quad ; \quad b_2 = \epsilon \sin \alpha(\epsilon) \quad (\epsilon \geq 0)$$

in (16) using its expression (15). Assuming α is at least twice continuously differentiable with respect to ϵ , we apply perturbation techniques to solve the equation

$$(17) \quad \frac{1}{2} \epsilon^2 \sin \alpha \cos \alpha - (1.5 - \sqrt{2}) \frac{(\cos \alpha - \beta^* \sin \alpha)^2}{(\cos^2 \alpha + 7 \cos \alpha \sin \alpha + 6 \sin^2 \alpha)} = 0$$

When $\epsilon = 0$, we have

$$\cot \alpha(0) = \beta^*$$

The derivative of the left hand side of (17) with respect to ϵ at $\epsilon = 0$ is zero. Differentiating twice with respect to ϵ and setting $\epsilon = 0$, we obtain

$$(18) \quad (2\sqrt{2} - 3) \frac{[\sin \alpha(0) + \beta^* \cos \alpha(0)]^2 \alpha_{\epsilon}^2(0)}{[\cos^2 \alpha(0) + 7 \cos \alpha(0) \cdot \sin \alpha(0) + 6 \sin^2 \alpha(0)]} + \sin \alpha(0) \cdot \cos \alpha(0) = 0,$$

or

$$(19) \quad \frac{\beta^*}{1 + (\beta^*)^2} - \frac{(3 - 2\sqrt{2})[1 + (\beta^*)^2]^2}{[(\beta^*)^2 + 7\beta^* + 6]} \alpha_{\epsilon}^2(0) = 0$$

so

$$(20) \quad \alpha_{\epsilon}(0) = \pm \left\{ \frac{(2\sqrt{2} + 3)[(\beta^*)^2 + 7\beta^* + 6]\beta^*}{[1 + (\beta^*)^2]^3} \right\}^{\frac{1}{2}}$$

$$\simeq \pm .083945$$

and therefore

$$(21) \quad b_1 = \epsilon \cos[\cot^{-1} \beta^* \pm .083945\epsilon + O(\epsilon^2)]$$

$$(22) \quad b_2 = \epsilon \sin[\cot^{-1} \beta^* \pm .083945\epsilon + O(\epsilon^2)]$$

The angle defined by the two curves given parametrically in (21, 22) contains the line $b_1 = \beta^* b_2$, showing that the double root β^* splits in two, different ones, thus, the destabilizing effect disappears not only at $b_1 = \beta^* b_2$, but whenever b_1 and b_2 are in that angle, or more precisely, in the cuspidal region defined by b_{21} and b_{22} .

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Captions for figures

Figure 1 Mechanical model.

Figure 2a Real part of the solution $\sigma(\lambda)$ of (7), vs the force λ ,
 $b_1 = .1, b_2 = 0$

Figure 2b Imaginary part of the solution $\sigma(\lambda)$ of (7), vs the force λ ,
 $b_1 = .1, b_2 = 0$.

Figure 2c Real and imaginary parts of the solution $\sigma(\lambda)$ of (7) corresponding
to figures 2a,b : $b_1 = .1, b_2 = 0$

Figure 3a Same as Figure 2a with $b_1 = .1, b_2 = .2$

Figure 3b Same as Figure 2b with $b_1 = .1, b_2 = .2$

Figure 3c Same as Figure 2c with $b_1 = .1, b_2 = .2$

Figure 4 Critical force λ_c vs. b_2 . $b_1 = 1.126$

Figure 5 Locus in the b_1 - b_2 plane where $\lambda_c = \lambda_c^u = 2$

Figure 6 The ratio λ_c/λ_c^u , λ_c and λ_c^u from reference 4, $b_1 = 1.125$

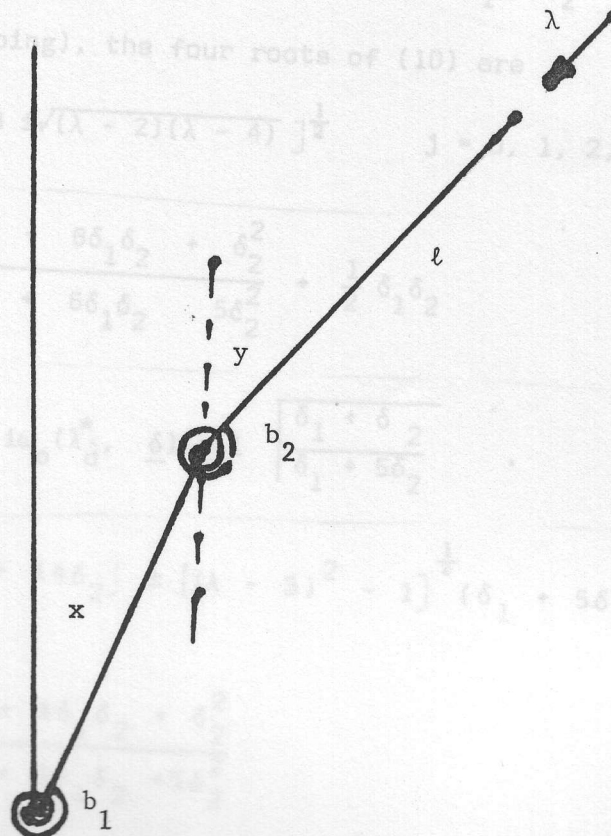


Figure 1

$$\lambda_d \sim \lambda_u + \frac{1}{D} \left(\sum_{k=1}^S \left[\sum_{j=1}^n (-1)^j \frac{\partial a}{\partial \delta_k} 2^{j-1} \omega_0^{2j-2} \right] \delta_k \right)^2$$

ith

$$D = \left(\sum_{k=1}^S \left[\sum_{j=1}^n (j-1)(-1)^j \frac{\partial a}{\partial \delta_k} 2^{j-1} \omega_0^{2j-4} \right] \delta_k \right)^2_B - \left(\sum_{k=1}^S \left[\sum_{j=1}^n (-1)^j \frac{\partial a}{\partial \delta_k} 2^{j-1} \omega_0^{2j-2} \right] \delta_k \right).$$

$$\left(\sum_{k=1}^S \left[\sum_{j=1}^n (-1)^j \left\{ 2 \frac{\partial^2 a}{\partial \lambda \partial \delta_k} 2^{j-1} \omega_0^{2j-2} + 2(j-1) \frac{\partial a}{\partial \delta_k} 2^{j-1} \omega_0^{2j-4} A + (j-1)(j-2) \frac{\partial a}{\partial \delta_k} 2^{j-1} \omega_0^{2j-6} B \right\} \right] \delta_k \right)$$

if $\omega_0 \neq 0$ and

$$\lambda \sim \lambda_u + \frac{\sum_k \frac{\partial a_1}{\partial \delta_k} \delta_u}{\sum_k \left[\frac{1}{a_2} \frac{\partial a_3}{\partial \delta_k} \frac{\partial a_0}{\partial \delta} - \frac{\partial^2 a_1}{\partial \delta_k \partial \lambda} \right] \delta_k} \quad \text{if } \omega_0 = 0.$$

$$(8) \quad 2x'' + y'' \cos(x-y) + (y')^2 \sin(x-y) + (\delta_1 + \delta_2)x' - \delta_2 y' + 2x-y - \lambda \sin(x-y) = 0$$

$$(9) \quad x'' \cos(x-y) + y'' - (x')^2 \sin(x-y) - \delta_2 x' + \delta_2 y' - x + y = 0$$

$$(10) \quad z^4 + (\delta_1 + 5\delta_2)z^3 + (6 + \delta_1\delta_2 - 2\lambda)z^2 + (\delta_1 + \delta_2)z + 1 = 0$$

If $\underline{\delta} = \underline{0}$, (no damping), the four roots of (10) are

$$z_j(\lambda) = \pm \left[\lambda - 3 \pm \sqrt{(\lambda - 2)(\lambda - 4)} \right]^{\frac{1}{2}} \quad j = 0, 1, 2, 3$$

$$(11) \quad \lambda_d^* = 2 \frac{\delta_1^2 + 6\delta_1\delta_2 + \delta_2^2}{\delta_1^2 + 6\delta_1\delta_2 + 5\delta_2^2} + \frac{1}{2} \delta_1\delta_2$$

$$\sigma_0(\lambda, \underline{\delta}) = i\omega_0(\lambda_d^*, \underline{\delta}) = i \sqrt{\frac{\delta_1 + \delta_2}{\delta_1 + 5\delta_2}}.$$

$$-[\delta_1 + 5\delta_2]\lambda + [2\delta_1 + 14\delta_2] \pm [(\lambda - 3)^2 - 1]^{\frac{1}{2}} (\delta_1 + 5\delta_2) = 0$$

$$\lambda_d = 2 \frac{\delta_1^2 + 6\delta_1\delta_2 + \delta_2^2}{\delta_1^2 + 6\delta_1\delta_2 + 5\delta_2^2}$$

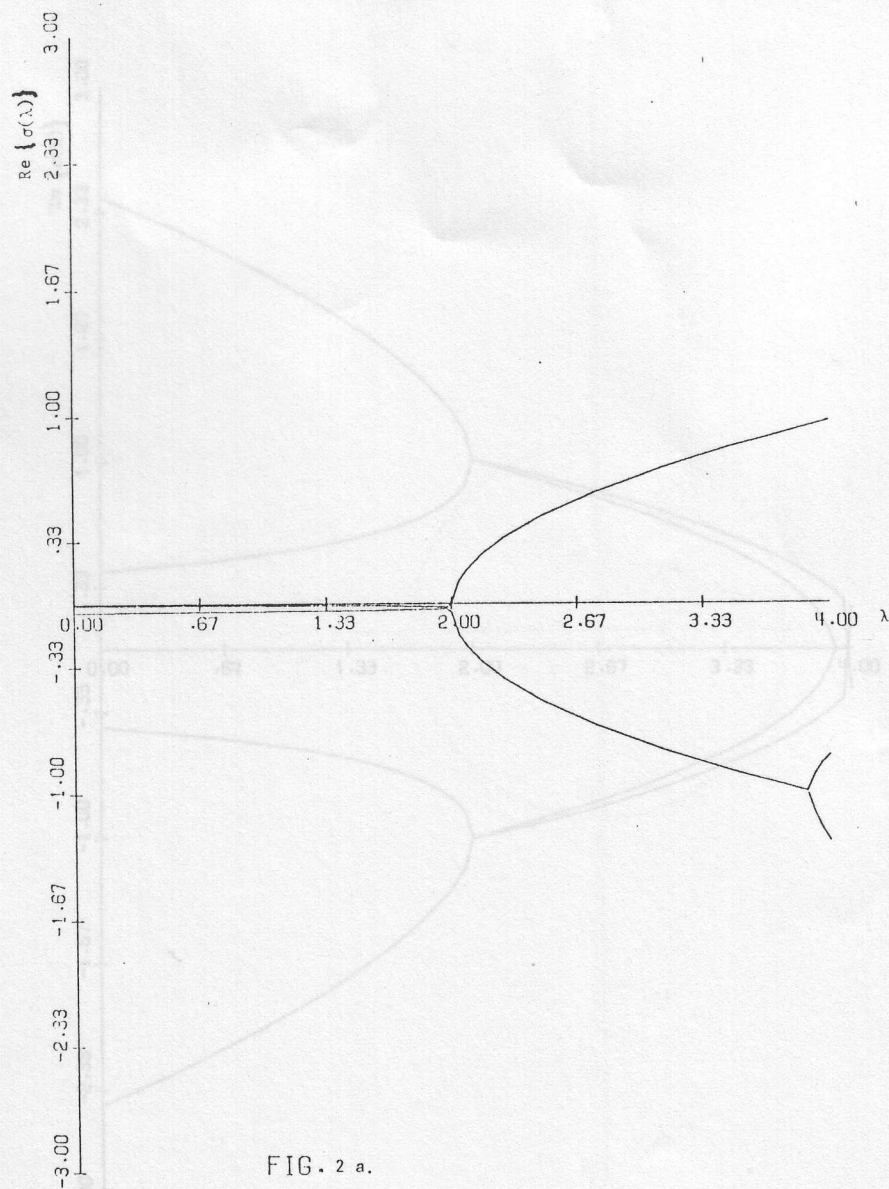


FIG. 2 a.

B1= .100

B2= 0.000

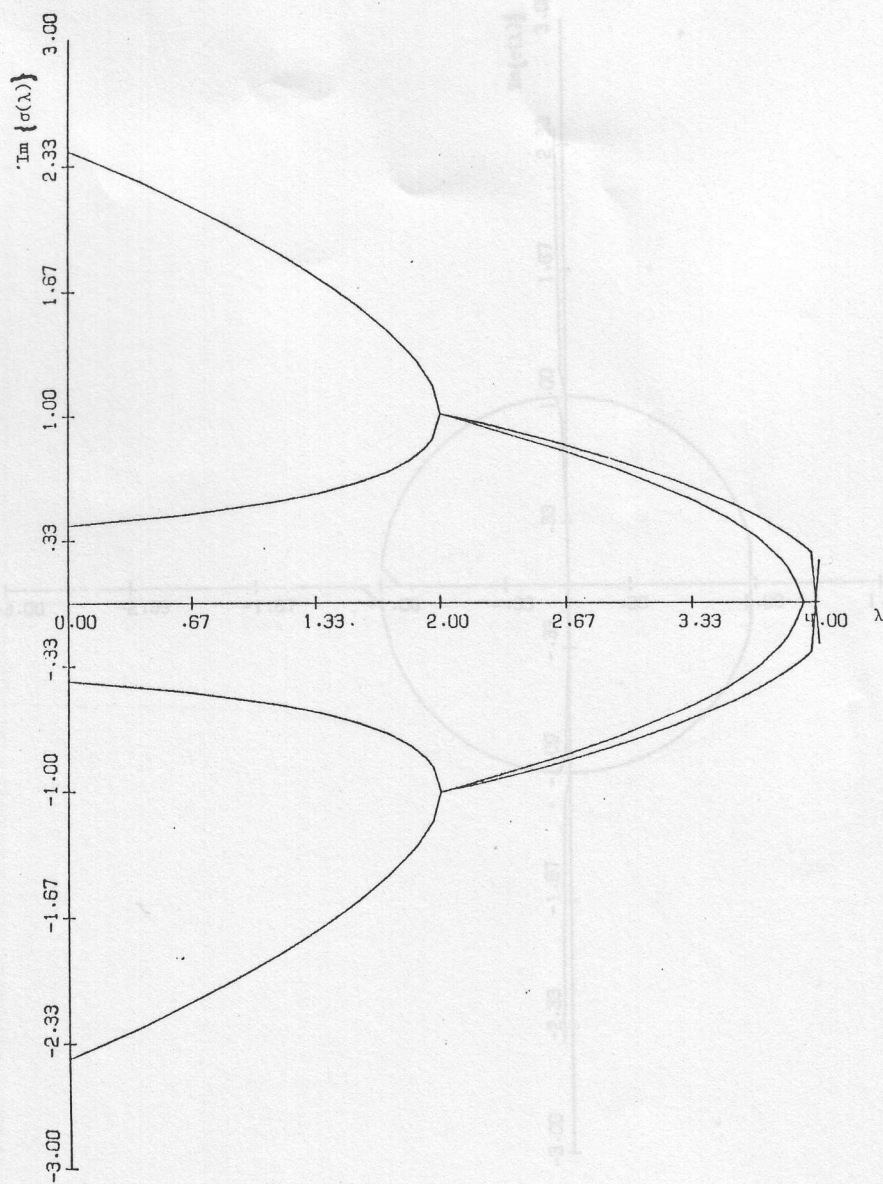


FIG. 2 b.

B1= .100

B2= 0.000

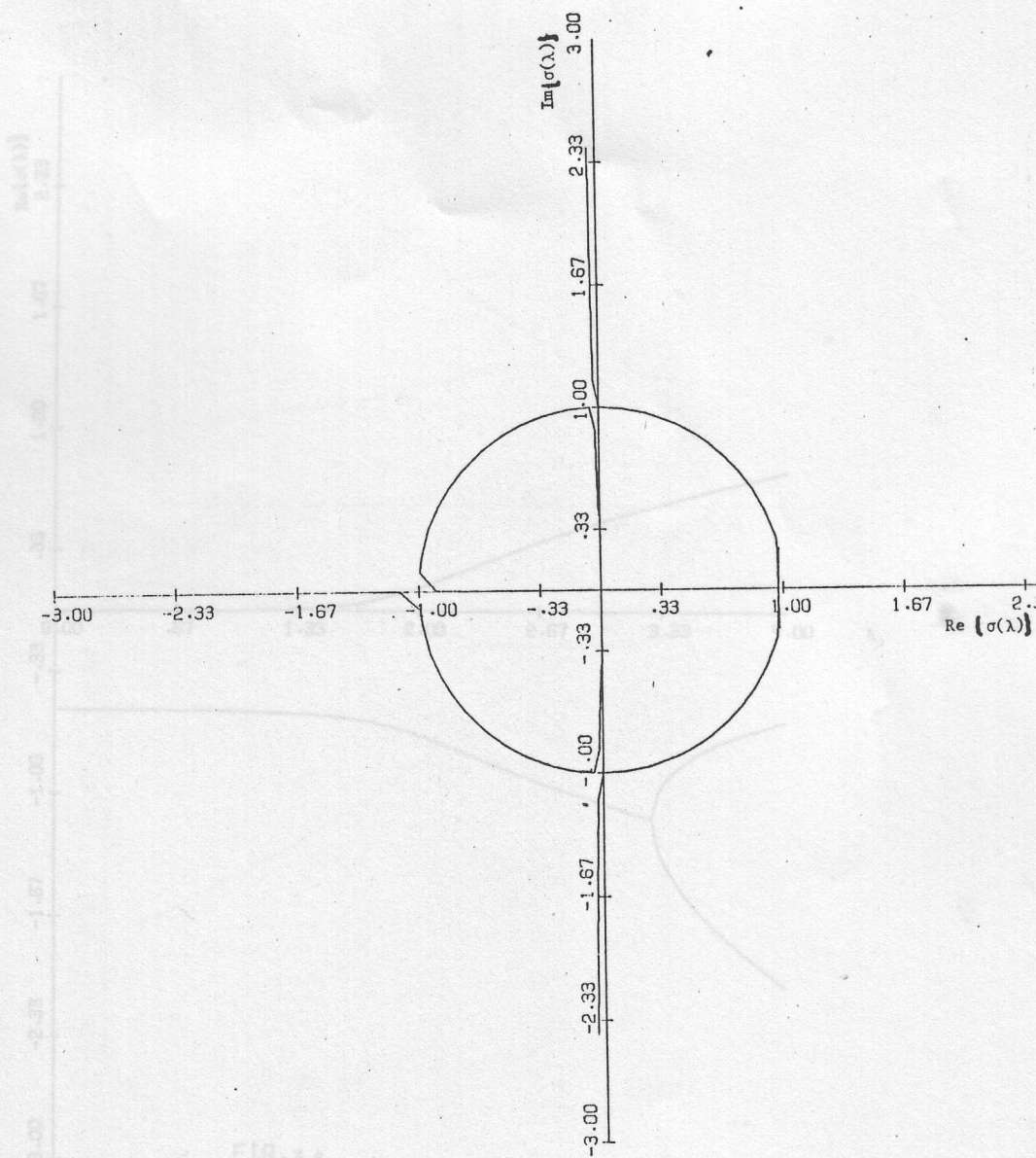


FIG. 2 c.

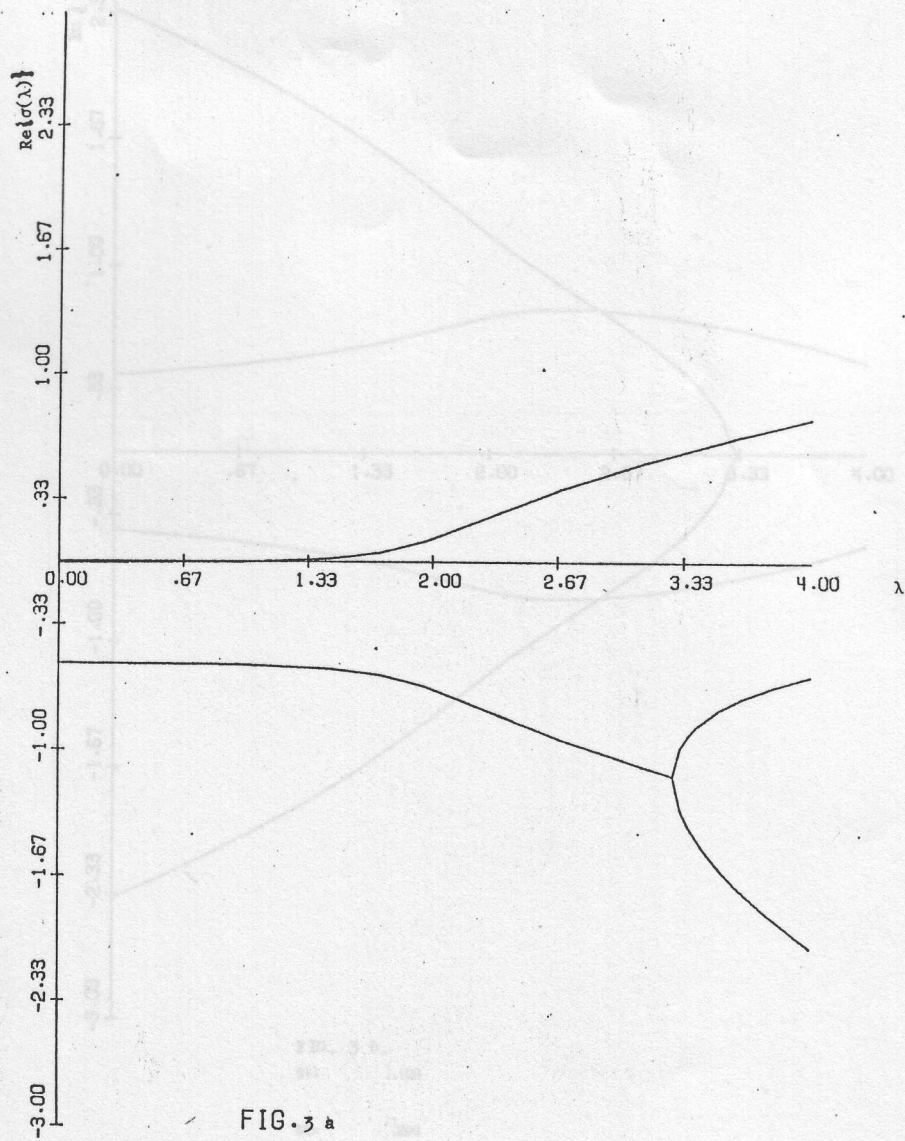


FIG. 3 a

B1= .100

B2= .200

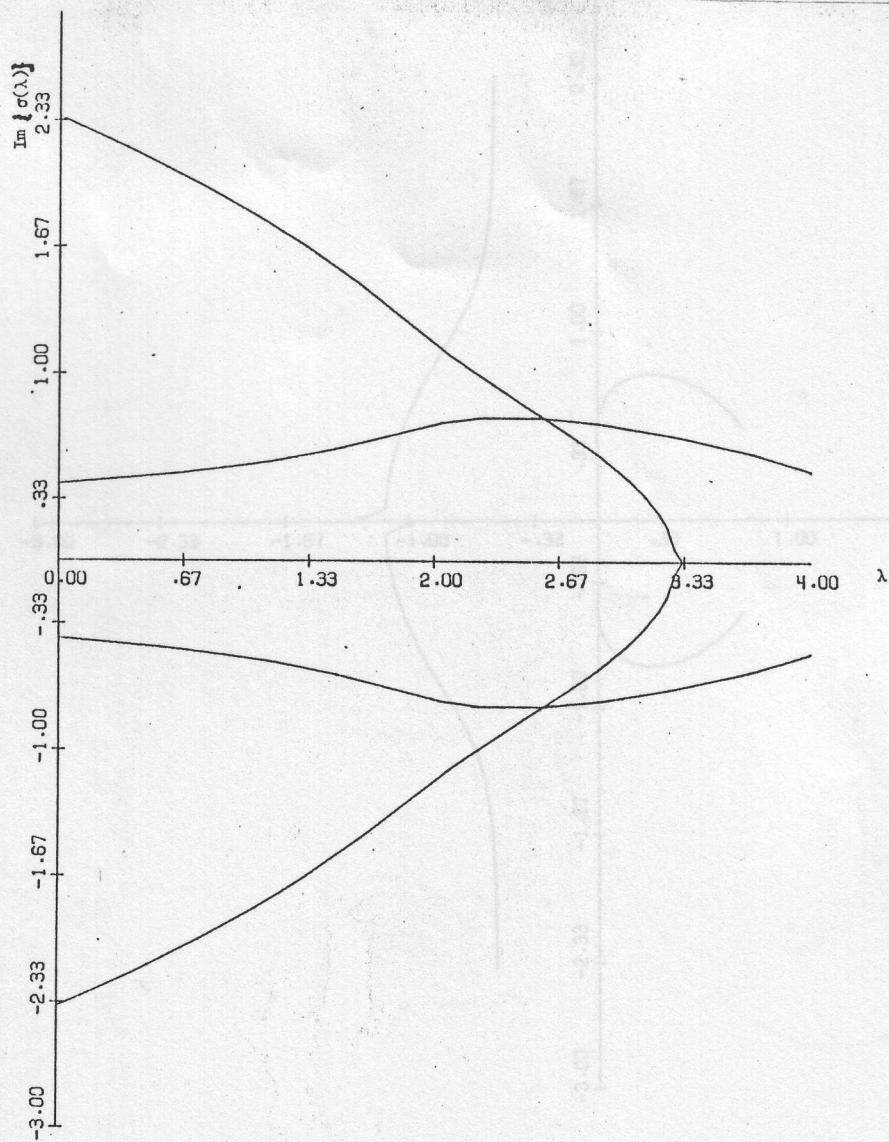


FIG. 3 b.

B1= .100

B2= .200

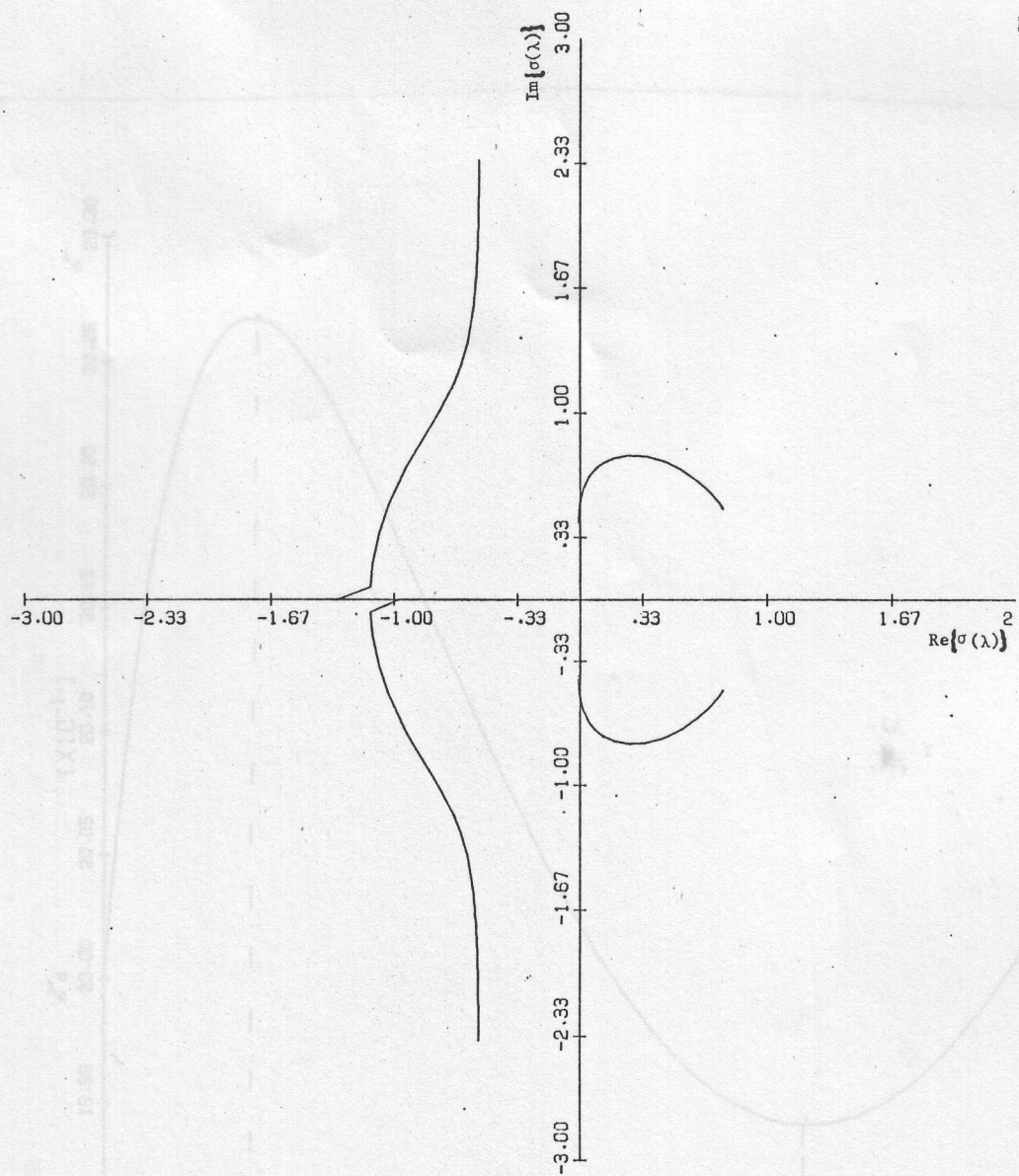


FIG. 3 c.

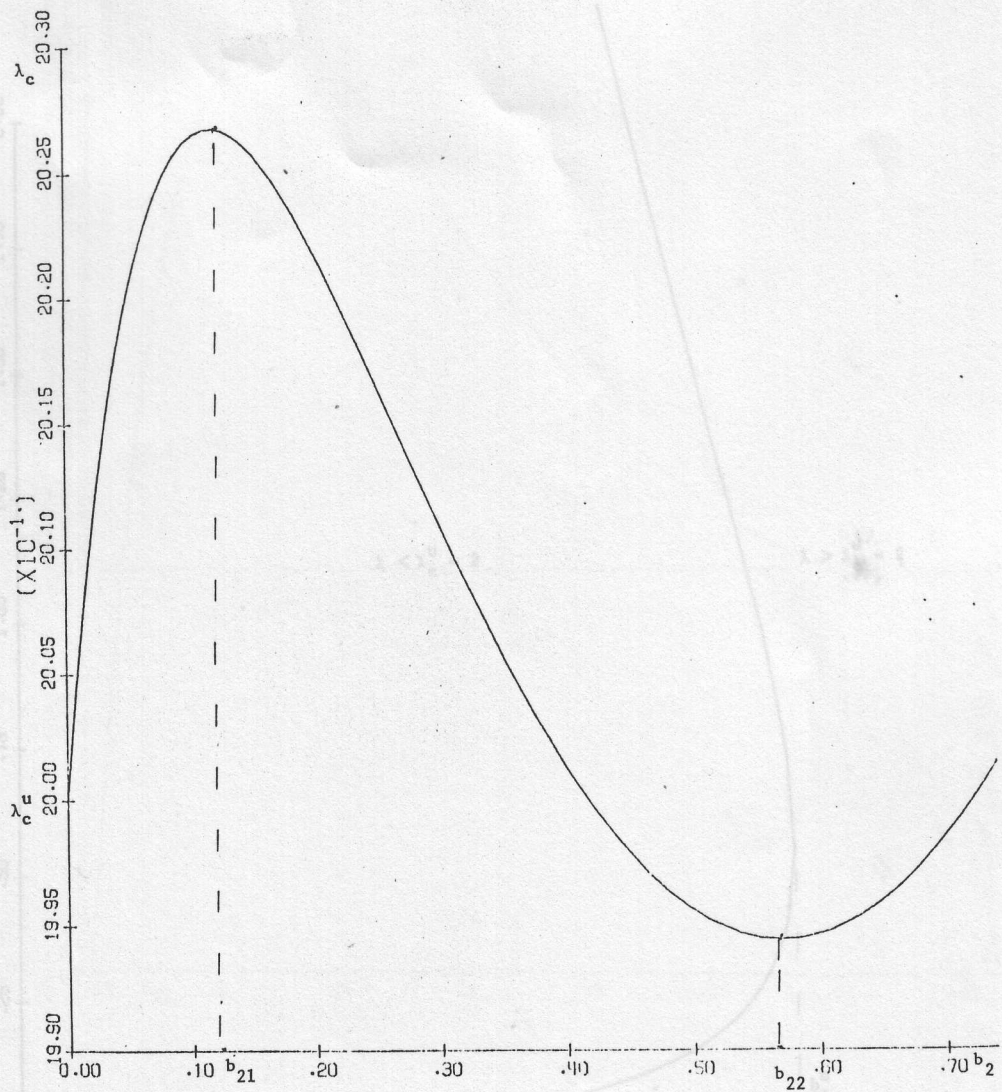


FIG. 4

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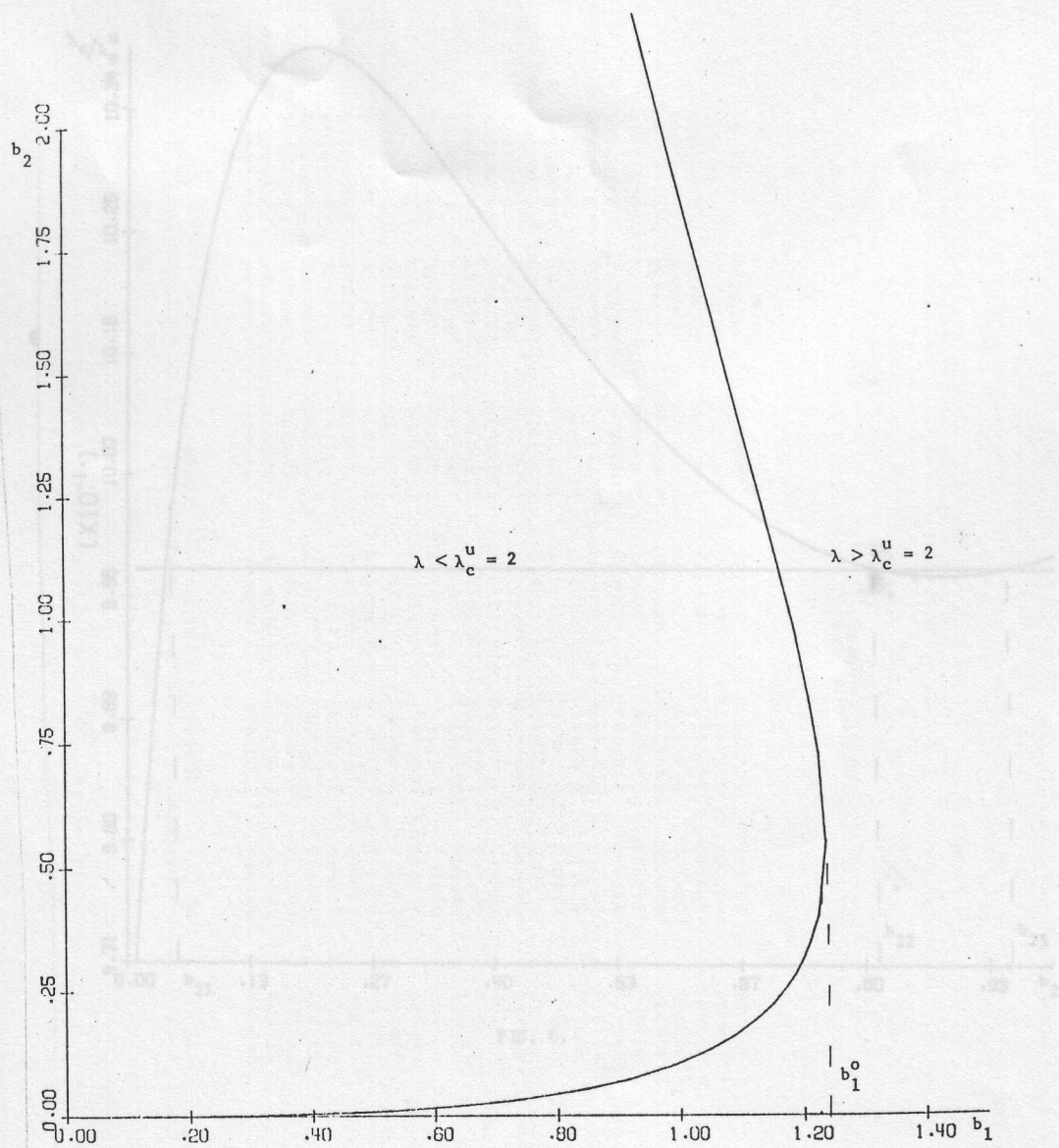


FIG. 5

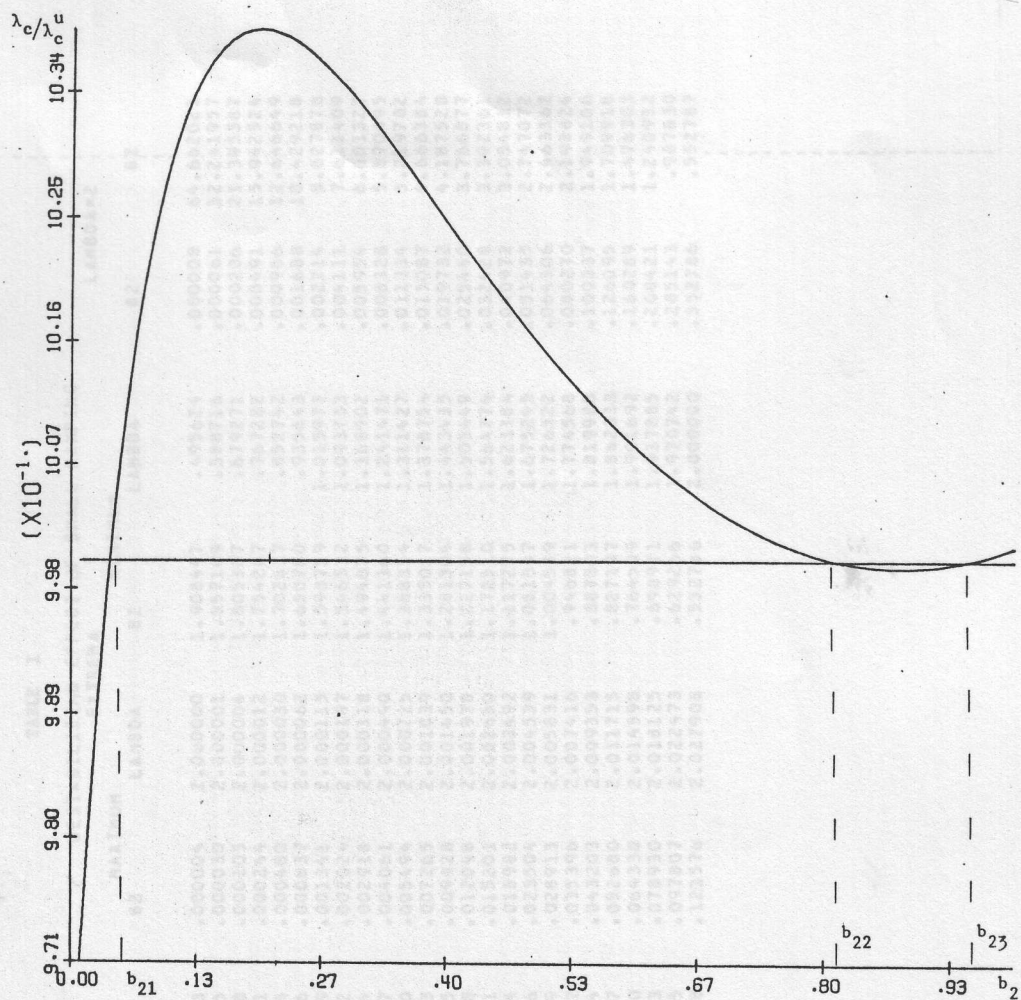


FIG. 6.

TABLE I
DESTABILIZING EFFECT OF SMALL DAMPING
EXTREMA

| B1 | MAXIMUM | | MINIMUM | | LAMBDA=2 | |
|----------|---------|----------|----------|----------|----------|-----------|
| | B2 | LAMBDA | B2 | LAMBDA | B2 | B2 |
| .049443 | .000004 | 2.000000 | 1.908447 | .495624 | .000008 | 64.662021 |
| .098885 | .000030 | 2.000001 | 1.857189 | .588716 | .000061 | 32.241957 |
| .148328 | .000103 | 2.000004 | 1.805307 | .679271 | .000206 | 21.395587 |
| .197771 | .000244 | 2.000012 | 1.754287 | .767282 | .000491 | 15.942524 |
| .247214 | .000480 | 2.000030 | 1.702617 | .852742 | .000966 | 12.646649 |
| .296656 | .000837 | 2.000062 | 1.650780 | .935643 | .001688 | 10.429218 |
| .346099 | .001341 | 2.000115 | 1.598759 | 1.015977 | .002714 | 8.827876 |
| .395542 | .002024 | 2.000197 | 1.546532 | 1.093733 | .004111 | 7.611409 |
| .444984 | .002918 | 2.000318 | 1.494075 | 1.168902 | .005954 | 6.661327 |
| .494427 | .004061 | 2.000490 | 1.441360 | 1.241471 | .008328 | 5.870495 |
| .543870 | .005494 | 2.000725 | 1.383354 | 1.311427 | .011334 | 5.219782 |
| .593313 | .007265 | 2.001039 | 1.335017 | 1.378754 | .015087 | 4.666384 |
| .642755 | .009428 | 2.001450 | 1.281304 | 1.443435 | .019732 | 4.187528 |
| .692198 | .012048 | 2.001978 | 1.227158 | 1.505449 | .025440 | 3.766877 |
| .741641 | .015201 | 2.002650 | 1.172510 | 1.564774 | .032428 | 3.392361 |
| .791084 | .016982 | 2.003492 | 1.117275 | 1.621384 | .040972 | 3.054812 |
| .840526 | .023504 | 2.004539 | 1.061347 | 1.675245 | .051435 | 2.747072 |
| .889969 | .028913 | 2.005831 | 1.004589 | 1.726322 | .064306 | 2.463362 |
| .939412 | .035396 | 2.007416 | .946821 | 1.774368 | .080270 | 2.198824 |
| .988854 | .043203 | 2.009353 | .887803 | 1.819928 | .100337 | 1.949106 |
| 1.038297 | .052680 | 2.011715 | .827197 | 1.862333 | .126095 | 1.709918 |
| 1.087740 | .064330 | 2.014598 | .764509 | 1.901692 | .160289 | 1.476303 |
| 1.137183 | .078930 | 2.018125 | .698971 | 1.937885 | .208421 | 1.240932 |
| 1.186625 | .097807 | 2.022473 | .629266 | 1.970742 | .285143 | .987630 |
| 1.236068 | .123576 | 2.027908 | .552786 | 2.000000 | .552786 | .552787 |

TABLE II

| DESTABILIZING EFFECT OF SMALL DAMPING | | | |
|---------------------------------------|-----------|-----------|----------------|
| B1 | B21 | B22 | B23 |
| .00010000 | .00009093 | .00000403 | 35049.06195114 |
| .00020000 | .00001806 | .00001807 | 17524.53077347 |
| .00030000 | .00002709 | .00002711 | 11683.02029110 |
| .00040000 | .00003612 | .00003614 | 8762.26498254 |
| .00050000 | .00004514 | .00004518 | 7009.81174352 |
| .00060000 | .00005416 | .00005423 | 5841.50923925 |
| .00070000 | .00006319 | .00006327 | 5007.00792629 |
| .00080000 | .00007221 | .00007231 | 4381.13188288 |
| .00090000 | .00008122 | .00008136 | 3894.33901918 |
| .00100000 | .00009024 | .00009041 | 3504.90486127 |
| .00110000 | .00009926 | .00009946 | 3186.27688939 |
| .00120000 | .00100827 | .00010831 | 2920.75355704 |
| .00130000 | .00111728 | .00011757 | 2696.07994740 |
| .00140000 | .00122629 | .00012662 | 2503.50254846 |
| .00150000 | .00133530 | .00013568 | 2336.60211808 |
| .00160000 | .00144430 | .00014474 | 2190.56422466 |
| .00170000 | .00155331 | .00015380 | 2061.70724402 |
| .00180000 | .00166231 | .00016286 | 1947.16769071 |
| .00190000 | .00177131 | .00017193 | 1844.68491830 |
| .00200000 | .00188031 | .00018099 | 1752.45040966 |
| .00210000 | .00198931 | .00019006 | 1669.00012710 |
| .00220000 | .00209831 | .00019913 | 1593.13622162 |
| .00230000 | .00220730 | .00020820 | 1523.86916577 |
| .00240000 | .00231629 | .00021727 | 1460.37435335 |
| .00250000 | .00242528 | .00022635 | 1401.95911514 |

→D()/DB1

| 1/B21+ | 1/B22+ |
|-------------|-------------|
| 11.07207160 | 11.07007396 |
| 11.07315104 | 11.06907541 |
| 11.07419274 | 11.06798197 |
| 11.07523349 | 11.06691844 |
| 11.07627630 | 11.06586701 |
| 11.07731817 | 11.06482170 |
| 11.07836009 | 11.06378508 |
| 11.07940206 | 11.06274060 |
| 11.08044409 | 11.06169688 |
| 11.08148618 | 11.06065553 |
| 11.08252832 | 11.05961596 |
| 11.08357052 | 11.05857480 |
| 11.08461277 | 11.05753384 |
| 11.08565508 | 11.05649308 |
| 11.08669744 | 11.05545133 |
| 11.08773986 | 11.05441103 |
| 11.08878234 | 11.05336988 |
| 11.08982487 | 11.05232903 |
| 11.09086745 | 11.05128846 |
| 11.09191009 | 11.05024818 |
| 11.09295279 | 11.04920773 |
| 11.09399554 | 11.04816717 |
| 11.09503835 | 11.04712692 |
| 11.09608121 | 11.04608660 |
| 11.09712413 | 11.04504660 |

Let $Q(\lambda, \underline{\delta}, z)$ be the $2n$ -th degree polynomial

$$Q(\lambda, \underline{\delta}, z) = \sum_{r=0}^{2n} a_r(\lambda, \underline{\delta}) z^r, \quad n \geq 2$$

$$P(\lambda, z) = Q(\lambda, \underline{0}, z) = \sum_{j=0}^n a_{2j}(\lambda, \underline{0}) z^{2j}.$$

$$(2) \quad \sigma_0(\lambda, \underline{\delta}) = -\frac{1}{2} \sum_{k=1}^s \left[\frac{\sum_{j=1}^n (-1)^j \frac{\partial a}{\partial \delta_k} 2j-1 \omega(\lambda)^{2j-1}}{\sum_{j=1}^n j(-1)^j a_{2j} \omega(\lambda)^{2j-1}} \right] \delta_k +$$

$$+ 1 \left\{ \omega(\lambda) - \frac{1}{2} \sum_{k=1}^s \left[\frac{\sum_{j=0}^n (-1)^j \frac{\partial a}{\partial \delta_k} 2j \omega(\lambda)^{2j}}{\sum_{j=1}^n j(-1)^j a_{2j} \omega(\lambda)^{2j-1}} \right] \delta_k \right\} + o(|\underline{\delta}|^2)$$

$$(3) \quad \sum_{k=1}^s \left[\frac{\sum_{j=1}^n (-1)^j \frac{\partial a}{\partial \delta_k} 2j-1 \omega(\lambda)^{2j-1}}{\sum_{j=1}^n j(-1)^j a_{2j} \omega(\lambda)^{2j-1}} \right] \delta_k = 0$$

$$(4) \quad \omega(\lambda) \sim \sqrt{\omega_0^2 + A(\lambda - \lambda_U) \pm \sqrt{\Delta}} \quad \text{when } \omega_0 \neq 0;$$

where

$$\Delta = B(\lambda - \lambda_U) + A^2(\lambda - \lambda_U)^2 + o(|\lambda - \lambda_U|)$$

$$A = -\frac{\sum_{j=1}^n j(-1)^j \frac{\partial a}{\partial \lambda} 2j \omega_0^{2j-2}}{\sum_{j=2}^n j(j-1)(-1)^j a_{2j} \omega_0^{2j-4}}$$

$$B = -2 \frac{\sum_{j=0}^n (-1)^j \frac{\partial a}{\partial \lambda} 2j \omega_0^{2j}}{\sum_{j=2}^n j(j-1)(-1)^j a_{2j} \omega_0^{2j-4}},$$

with $P(\lambda, z_0(\lambda)) = o(|\lambda - \lambda_U|)$, and

$$\omega(\lambda) \sim \sqrt{\frac{a_2}{2a_4} - \frac{\text{sgn} a_2}{2a_4} \sqrt{a_2^2 - 4a_4 \frac{\partial a_0}{\partial \lambda} (\lambda - \lambda_U)}},$$

with $p(\lambda, z_0(\lambda)) = o(|\lambda - \lambda_U|^2)$ if $\omega_0 = 0$.